


Static Surface Forces

- Forces on plane areas
- Forces on curved surfaces
- Buoyant force





So, this is a fluid statics 2, I mean, I call it 2, because we are going to do the surface forces and body forces therefore we need to know, what are we going to study in fluid statics 2. So, in statics 2 its static surface forces, we are going to see force on plane areas, okay. We are going to see force on curved surfaces, like this. So, this is plane areas, this is plane, this is curved surface and we also will see the buoyant force, a very small detail of it, but I think this is very necessary. So, I mean, if we are teaching basics of fluid mechanics in this course in the beginning, so, buoyancy is an important concept that everybody must be aware of. Good.

(Refer Slide Time: 15:47)

Forces on Plane Areas: Horizontal surfaces

net P = 500 kPa

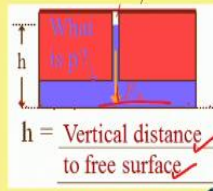
What is the force on the bottom of this tank of water?

$$F_R = \int p dA = p \int dA = pA \quad \text{gauge} \quad p = \rho gh$$

$$F_R = \rho g hA = \text{volume} \times \rho g$$

$F_R = \text{weight of overlying fluid}$

F is normal to the surface and towards the surface if p is positive.
F passes through the centroid of the area.

$$-\nabla p = \rho \mathbf{a} \quad \frac{\partial p}{\partial x} = \rho a_x = 0$$


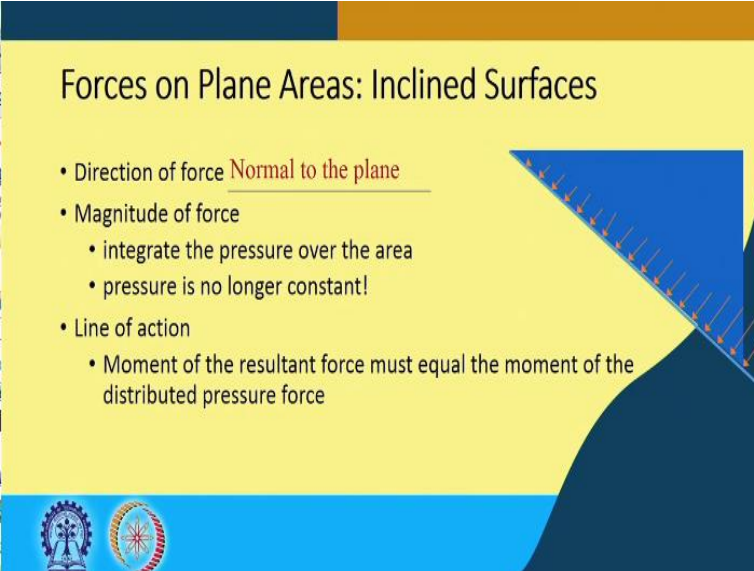
So, we have to see what are the forces on plane areas, that is horizontal surface. So, if you see, this is a figure that shows, you know, a horizontal surface a depth h, okay. So, this h is the

vertical distance to free surface and this we what is the P here, okay. And what is the resultant force at the bottom, okay, and P we are assuming 500 kilo Pascal's, okay, that we are going to see. So, what is the force on the bottom of this tank of water actually, what is the net force on the bottom of this tank?

So, the force resultant force is going to be the integration of pressure into area, So, p is constant so it comes out and that becomes pressure into area pA, where p is rho gh, okay, this is the gauge pressure. So, $F_R = \int p dA = p \int dA = pA$
 So, F R is actually nothing but the weight of the overlying fluid, okay. Also F is normal to the surface and towards the surface, if p is positive, okay.

F passes through the centroid of the area. This is an important information for you. And therefore, the change in pressure can be equated to rho into a, okay, or we can write in x direction $-\frac{\partial p}{\partial x} = \rho a_x = 0$
 So, there is no acceleration in x direction here, and that is equal to 0. There is no pressure variation in x direction whatever there is, it is in z direction. Good.

(Refer Slide Time: 18:08)



Forces on Plane Areas: Inclined Surfaces

- Direction of force Normal to the plane
- Magnitude of force
 - integrate the pressure over the area
 - pressure is no longer constant!
- Line of action
 - Moment of the resultant force must equal the moment of the distributed pressure force

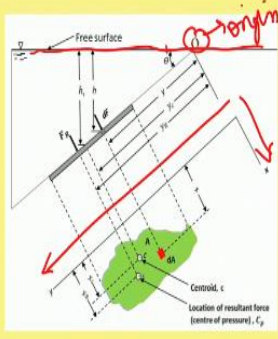
The slide includes a diagram of a blue fluid with a yellow inclined surface. Red arrows representing pressure forces act perpendicular to the surface. The bottom of the slide features two institutional logos.

Another important thing is, we have to learn and revise again, what are the forces on the plane areas or the inclined surface. So, this has to be taken in a little bit of more detail. What will be the direction of the force? Always perpendicular, normal to the plane, right. So, the force will

start acting like this, correct. What will be the magnitude of the force? We have to integrate the pressure over the entire area. Here, the pressure is no longer constant, because it is not at one elevation it is varying see, the h is changing here, here it is different, here it is different, here it is different.

So, what is the line of the action? So far to find the line of action, we have to do the moment of the resultant force must be equal to the moment of the distributed pressure force. We have to do the moment balance to find the line of action we will see soon how are we going to tackle, that so I will just erase all ink on the slide. Good.

(Refer Slide Time: 19:24)



Forces on Plane Areas: Inclined Surfaces

Determine location, direction and magnitude of the Resultant force acting on one side of this area due to the liquid in contact with water

Let the plane in which the surface lies intersect the free surface at point O

Let this make an angle θ with the surface

The x-y coordinate system is defined such that O is the origin and x axis is directed along the surface as shown

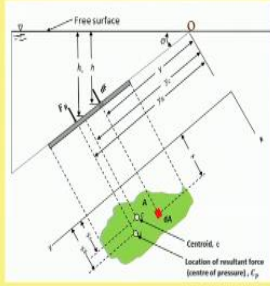
So, forces on plane areas, so this has been taken actually from Munson, Young and Okkiishi the derivation. So, but I think I will explain one by one what those things are, So, the question, the biggest question is you have to determine the location direction and magnitude of the resultant force acting on one side of this area due to the liquid in contact with water. If you see the body, okay. I will just erase because this was just to you know, okay, alright.

What we say, let the plane in which the surface lies, intersect the free surface. So, this is the free surface here, okay, and let the plane in which the surface lies the body intersect at point O, okay, right. Good. And let this make an angle θ with the surface, right. The xy coordinate system is defined such that O is the origin. So, this is also the origin, you know, and the x axis is directed

along the surface as shown so this is x axis, okay. Sorry, this is x axis and this is y axis, good. So, this is just explaining you, and we are going to look at the individual terms when we describe those, you see many things here, centroid, location of resultant forces, you see this F_R , you see dF , but we will come to it one by one, good.

(Refer Slide Time: 21:15)

Forces on Plane Areas: Inclined Surfaces



At any given depth h , Force acting on dA

$$dF = \gamma h dA \quad (\text{perpendicular to surface})$$

$$F_R = \int_A \gamma h dA = \int_A \gamma y \sin \theta dA$$

$$F_R = \gamma \sin \theta \int_A y dA$$

First moment of inertia

$$\int_A y dA = y_c A$$

y_c is coordinate of centroid of area A measured from x axis which passes through O .

$$F_R = \gamma A y_c \sin \theta$$

$F_R = \gamma A h_c$

h_c vertical distance from fluid surface to centroid of area

So, what I have done is, I have kept this image on the left side and how we are going to. So, for any depth h , okay. Let us say that the force acting on an area dA will be because it is at depth h , dF will be pressure into area right $\gamma h dA$, this is the dF , this is perpendicular to the surface, that is very important. So, to find out the resultant force what should we do? We do $\gamma h dA$ integrated over the entire area, right.

h here, if we start if we try to write down in terms of Y this can be related to θ as h is $y \sin \theta$ and γ is γ and dA is dA , okay. So, F_R can be written as $\gamma \sin \theta$ can come out and this is $y dA$, okay, integral of $y dA$, very important, if you remember from your earlier classes, this is first moment of inertia. And this integral $y dA$ can be written as y_c into A , where y_c is the location of the centroid of the object, right. And F_R can be written as

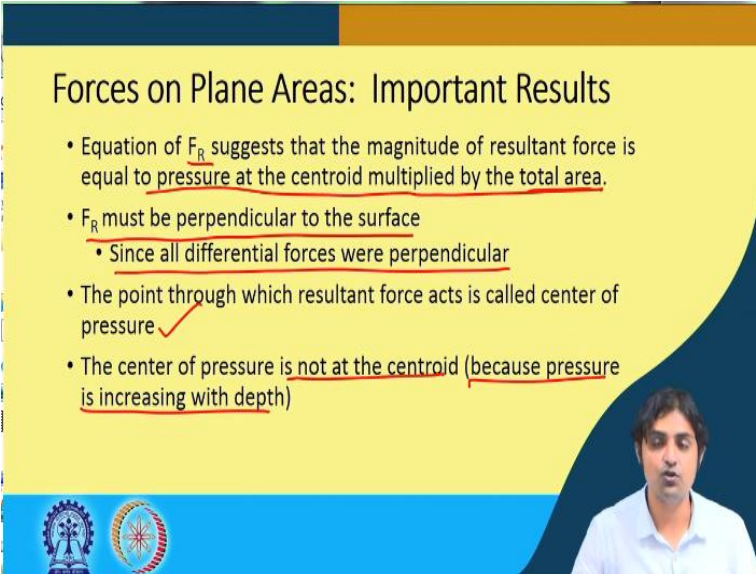
$$F_R = \gamma A y_c \sin \theta$$

. So now, you know γ , you know A , you know $\sin \theta$ and for a particular object you also know y_c , the centroid location, right. Where y_c is the coordinate of the centroid of the area A measured from x -axis which passes through O . So, orientation you have to be very careful about,

okay. I will just erase all the ink now. So, or we can also say γA from here and $y_c \sin \theta$. What is $y_c \sin \theta$? This is y_c , right? $y_c \sin \theta$ can be written as h_c , h_c is the height of the centroid from the free surface.

So, h_c is the vertical distance from fluid surface to the centroid of the area. Now, we have actually simplified into very common, I mean, very simple equation. I will erase, so I will just write the important one, so this is the important one, F_R is equal to $\gamma A h_c$ this is one important result to note down at this point in time. Very good.

(Refer Slide Time: 23:57)



Forces on Plane Areas: Important Results

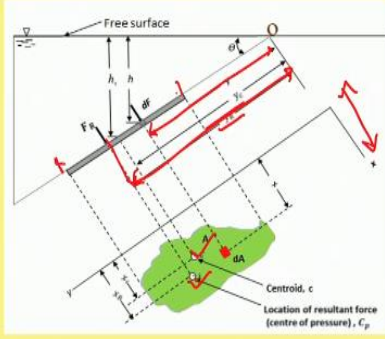
- Equation of F_R suggests that the magnitude of resultant force is equal to pressure at the centroid multiplied by the total area.
- F_R must be perpendicular to the surface
 - Since all differential forces were perpendicular
- The point through which resultant force acts is called center of pressure ✓
- The center of pressure is not at the centroid (because pressure is increasing with depth)

So, we proceed to the next slide, okay. So, this equation of F_R suggests, that the magnitude of the resultant force is equal to the pressure at the centroid multiplied by total area. As you have seen here, see, pressure γA multiplied by the total area, right. F_R also must be perpendicular to the surface that is very important. And the reason is since all the differential forces were perpendicular that we have counted.

If you see, all the forces these were all perpendicular. So, this is let us say dF_1 , dF_2 , dF_3 . So, if you add this the sum can change, but not the direction, right. The point through which the resultant force acts is called the center of pressure. This is you might have heard in your fluid mechanics class what calculate the center of pressure. So, this is what the center of pressure is a quick revision for you again. The center of pressure is not at the centroid. And what is the

reason? Because the pressure is increasing with depth, that is important again, another important result to note down, okay. We have seen the resultant force value and we have discussed some of the properties of resultant force. Now, we must also be able to find out what corresponding x R (Refer Slide Time: 25:35)

Center of Pressure: y_R



Coordinate y_r can be Determined by summation of moment around x-axis

and y_R is. So, let us say there is the one of the coordinates in y direction for the center of pressure centroid is here this is center of pressure right is y_R , which is not equal to y_c , okay. In that particular case what we are going to do is, coordinate y_R can be determined by summation of moment around x axis. So, for finding y_R the moment equilibrium should be done around x axis.

(Refer Slide Time: 26:15)

Center of Pressure: y_R

$$y_R F_R = \int_A y dF = \int_A \gamma \sin \theta y^2 dA \quad \text{Sum of the moments}$$

$$F_R = \gamma A y_c \sin \theta$$

$$y_R = \frac{\int_A y^2 dA}{y_c A} \quad \text{Second moment of Inertia } I_x \text{ wrt } x \text{ axis } y_R = \frac{I_x}{y_c A}$$

Using parallel axis theorem $I_x = I_{xc} + A y_c^2$

I_{xc} is the second moment of area wrt an axis passing through centroid and parallel to x axis

$$y_R = y_c + \frac{I_{xc}}{y_c A}$$

So, the way we do it $y R$, $F R$. So, $c y R$ into $F R$, so this is $y r$ into $F R$, this will be equal to integral $y dF$, right. So, what is $y dF$? So this is dF and this is y , so all the summation beginning from here, until here, all these small small dF that is there. So, that will be y can be written as, sorry, dF can be written as $\gamma \sin \theta y dA$, correct. This we have already seen in the previous slide this is the sum of the moment, where $F R$ was $\gamma A y c \sin \theta$ that we have seen before. So, $y R$ now can be written as, integral $y^2 dA$ divided by $y c$ into A . I will erase the ink.

So, you can see integral $y^2 dA$, okay, divided by $y c$ into A , because $\gamma \sin \theta$ $\gamma \sin \theta$ cancels, right. So, this is actually the second moment of inertia with respect to the x axis, very good. So, we are coming at some conclusions now, where $y R$ can be written as I_x second moment of inertia, correct. Now, if we use the parallel axis theorem. So, because see this is around an x axis which is not fully independent of the coordinate center, right system. But if we have a coordinate system passing through the centroid of the object, then we are able to you know, calculate very easily does not matter how do we orient our coordinate system.

So using parallel axis theorem I_x can be written as $I_x = I_{xc} + A y c^2$, very simple, that, this you have read before as well. So, I_{xc} is the second moment of inertia with respect to an axis passing through the centroid and parallel to x axis, therefore, the equation of $y R$ can be written as $y c$, is the centroid plus I_{xc} divided by $y c$ into A , I_{xc} is known, $y c$ is known and A is known, right.

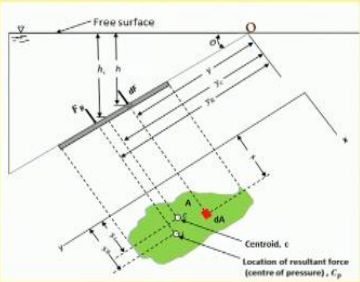
So, this is an important let me, so, this is the important equation again. So,

$$y_R = y_c + \frac{I_{xc}}{y_c A}$$

. If you remember this equation you will always be able to find out the center of pressure coordinate y_R .

(Refer Slide Time: 29:11)

Center of Pressure: x_R



Coordinate x_R can be determined by summation of moment around y-axis

So, we also need to find out x_R . What is the logical way? We will do the summation of moment around y axis now. So, for x_R we need to do the moment calculation around y axis so around this axis.

(Refer Slide Time: 29:29)

Center of Pressure: x_R

$$x_R F_R = \int_A x dF = \int_A \gamma \sin\theta xy dA \quad \text{Sum of the moments}$$

$$F_R = \gamma A y_c \sin\theta$$

$$x_R = \frac{\int_A xy dA}{y_c A} \quad \text{Product of Inertia } I_{xy} \text{ wrt } x \text{ and } y \text{ axis} \quad x_R = \frac{I_{xy}}{y_c A}$$

Using parallel axis theorem $I_{xv} = I_{xvc} + A y_c x_c$

I_{xyc} is the product of inertia wrt an orthogonal coordinate System passing through centroid

$$x_R = x_c + \frac{I_{xyc}}{y_c A} \quad \checkmark$$

How? See, X_R into F_R . Where is X_R ? This is X_R . So, X_R into F_R , okay. Because, yeah, is equal to integral $x dF$, so dF , we already know. So, it is $\gamma \sin \theta y dA$, right, and this is some of the moment we know, F_R is this one, $\gamma A y_c \sin \theta$. Therefore, x_R will come out to be integral $x y dA / y_c A$, and this is the product of inertia I_{xy} with respect to the x and y axis.

Therefore, x_R can be written as $I_{xy} / y_c A$. Again the same thing happens that we need to translate this moment of inertia or product of inertia I_{xy} with respect to the centroid, so that it becomes independent of that particular coordinate system. So, we can write, if this is I_{xy} , okay, I_{xy} can be written as I_{xy} is equal to I_{xy} at centroid plus $A y_c x_c$. If we put this in this equation, here I_{xy} is the product of inertia with respect to an orthogonal coordinate system passing through the centroid.

And therefore


$$x_R = x_c + \frac{I_{xyc}}{y_c A}$$

. So, this is an important result here, this is good. Another important result for we have obtained y_R and x_R .

(Refer Slide Time: 31:29)

Center of Pressure

- If the submerged area is symmetrical wrt to axis passing through centroid and parallel to either x or y axis, the resultant force must pass lie along $x=x_c$ since I_{xyc} is identically zero.
- As y_c increases, center of pressure moves closer to the centroid of the area.
- Since $y_c = h_c / \sin \theta$, y_c will increase if depth of submergence h_c increases or for a given depth the area is rotated such that the angle θ decreases.



Now, the center of pressure sum. If the submerged area is symmetrical with respect to axis passing through centroid and parallel to either x or y axis the resultant force must pass a lie along

the x is equal to x_c since I_{xy} is identically 0. See, if the submerged area is symmetrical with respect to axis passing through the centroid, then I_{xy} is identically 0. So, x_r will be 0. Now, as the y_c increases the center of the y coordinate of the centroid increases, center of pressure moves closer to the centroid of the area, very, very it is easy to see from the equation that y_R was $y_c +$.

You know if y_c is too large then does not matter what the other term is. Since, y_c is equal to h_c by $\sin \theta$, y will increase if depth of submergence h_c increases or for a given depth the area is rotated such that an angle- θ decreases. This is very obvious from this particular equation.

(Refer Slide Time: 32:36)

Properties of Areas

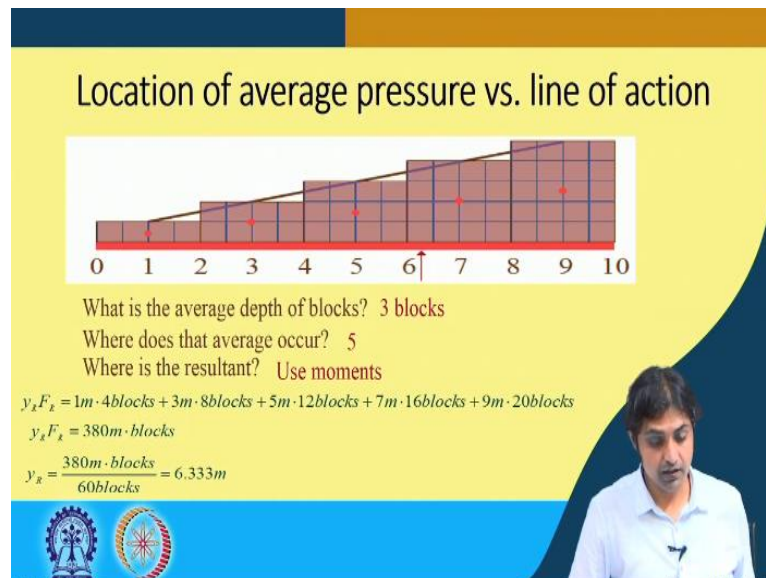
	$A = \frac{\pi R^2}{2}$ $y_c = \frac{4R}{3\pi}$ $I_{xc} = \frac{\pi R^4}{8}$ $I_{xy} = 0$ $\frac{I_{xc}}{A} = \frac{R^2}{4}$
	$A = \pi ab$ $y_c = a$ $I_{xc} = \frac{\pi ba^3}{4}$ $I_{xy} = 0$ $\frac{I_{xc}}{A} = \frac{a^2}{4}$
	$A = \frac{\pi R^2}{4}$ $y_c = \frac{4R}{3\pi}$ $I_{xc} = \frac{\pi R^4}{16}$ $\frac{I_{xc}}{A} = \frac{R^2}{4}$

So now, there are some properties of area, which you will be supplied with. This is very easy to know, hand on, you do not need to remember some other with this slide if you had, so area in this particular of rectangle area is $a b$, depth of the centroid is a by 2 , I_{xc} is $b a^3$ by 12 . So, this is symmetric about the centroid, right. So, I_{xy} is 0 , where I_{xc} by A is a square by 12 , I mean you can just simply see. Similarly, for this triangle, isosceles triangle, this is for circle, I think, you can just know, note it down these figures are quite important.

The circle for example, even objects like this because in fluid mechanics in hydraulics you will encounter structures like this having this type of gate, ellipse is also important. So area is $\pi a b$, y_c is a , I_{xc} is $\pi b a^3$ by 4 , I_{xy} is 0 , because it is also symmetric around the centroid. Similarly, this here, so this y_c is $4R$ by 3π , this value is important because, we are going to use

it because most of the gates that are used in hydraulics have these type of openings we will come and we will see in this type of you know thing.

(Refer Slide Time: 33:55)



So, this is the probably the last slide of this lecture I would want to each and every one of you, this a very basic question, which I have taken from an English author, location of average pressure versus line of actions. We will see in practice, how this moment and those things are calculated with the help of these blocks. This is quite easy to understand. So, my question is what is the average depth of the blocks? Anyone tell me what is the average depth? So, this is 1, 2, 3, 4, 5.

So, what is the average depth 3, right? Where does this average occur tell me? Where is this average occurring the depth tell me? So, it is happening at, see, at 5. Are you able to get this, right? See, this is, correct, to see that most number of the block number 3 is at 5, you know, 1 2 3 4 5 6 6 8 9 10. So if you take the depth. Now the most important, where is the resultant actually? How are you able to find this resultant? We will use moment balance here.

So, correct? So, what we are going to do? We are going to do y_R into F_R , resultant force is equal to 1 meter. So, this is 1 meter. How many blocks are there, tell me? 4 blocks plus, see, 1, 2 3, 4, okay? Now, 8 blocks into 3 meters, right, 1 2 3 4 5 6 7 8 into 3 meters. Now we have 12 blocks, 1, 2, 3, 4, 5, 6, 7, 8, 9, 10, 11, 12, 12 blocks that 1. Similarly, we have 16 blocks we have

20 blocks and if we do this summation so $y R F R$ is equal to 380 meter blocks, okay. And therefore, 380 meter blocks in the total number of blocks is 60 blocks.

So, we divide and we see y_R will be 6.333 meters. So, this is quite an intriguing example, where you can practice it on your own. And if you do not understand, we will explain that in the forum if you please, raise your questions there, that is good. So now, I will end this particular lecture and we will proceed to the last lecture of our this week which is continuation of fluid statics where we will see some more examples and how the forces on the surfaces is calculated more, great, see you.